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ATERAL HEAT FLOW EFFECTS ON THERMOGRAPHIC SENSITIVITY

Ignacio Perez, Paul Kulowitch NAVAL AIR WARFARE CENTER AIRCRAFT DIVISION, PATUXENT RIVER, MD

THE SECOND JOINT NASA/FAA/DoD CONFERENCE ON AGING AIRCRAFT

August 31 - September 3, 1998

Williamsburg Marriott Hotel Williamsburg VA

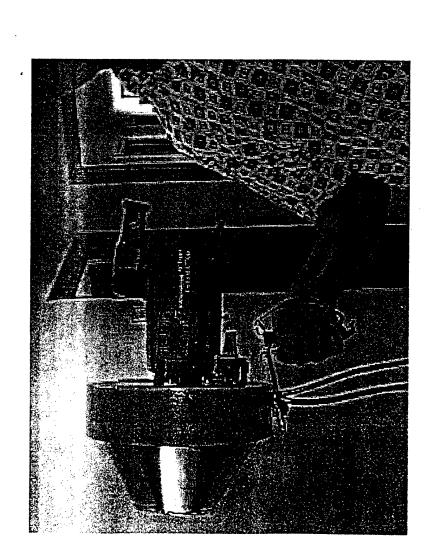
CLEARED FOR OPEN PUBLICATION

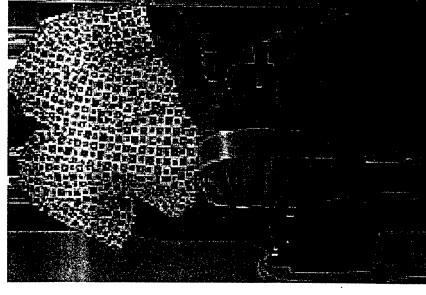
PUBLIC AFFAIRS OFFICE NAVAL AIR SYSTEMS COMMAND



PORTABLE IR CAMERA SYSTEM







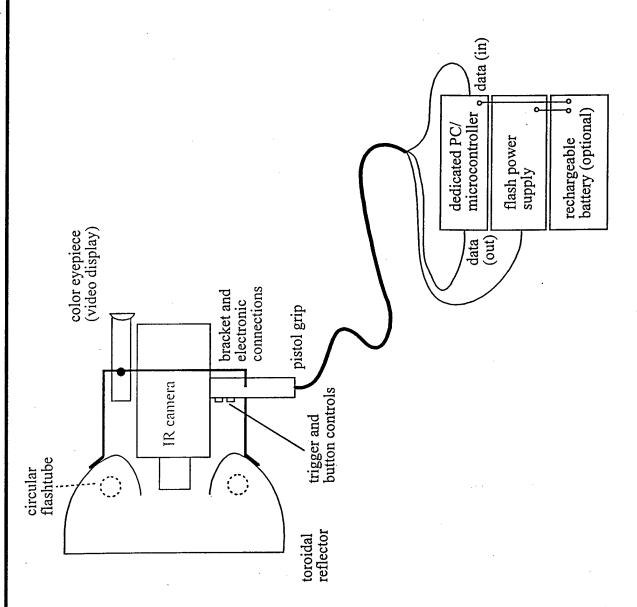
AND POWER SUPPLY

CAMERA HEAD



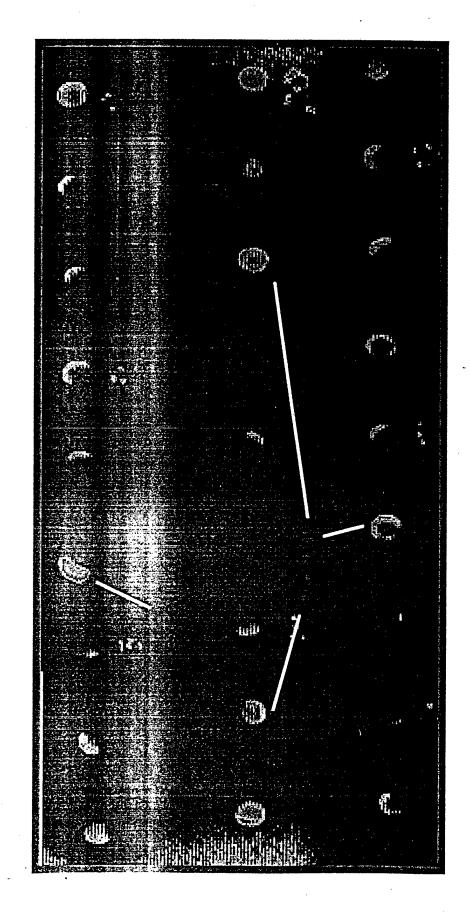
PORTABLE IR CAMERA SYSTEM











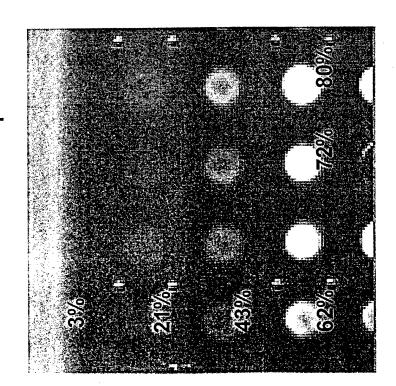




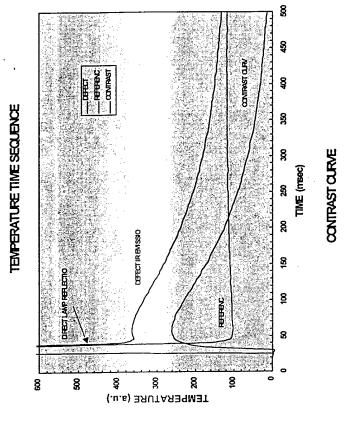
TEST PANEL & TYPICAL TIME-RESPONSE CURVES

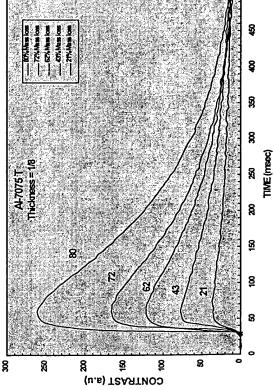


1/8" Thick AI-7075 panel



1" Diameter Holes







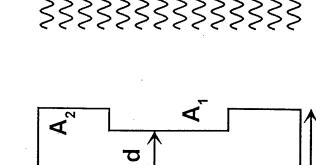
NO LATERAL HEAT CONDUCTIVITY APPROXIMATION

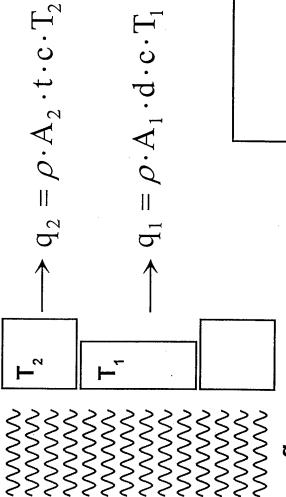


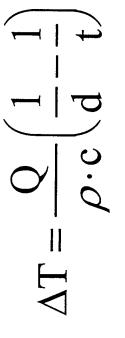
FLAT BOTTOM HOLE

NO LATERAL CONDUCTION APPROXIMATION

 $q = m \cdot c \cdot \Delta T$







$$\Delta T = T_1 - T_2$$

Q=q/A



CONTRAST PROPERTIES



$$\Delta T = \frac{Q}{\rho \cdot c} \left(\frac{1}{d} - \frac{1}{t} \right)$$

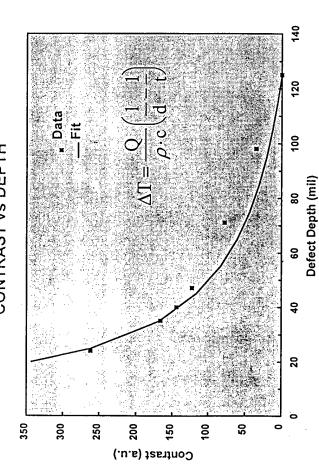
- OF 1. THE CONTRAST (AT) INCREASES LINEARLY WITH THE AMOUNT DEPOSITED ENERGY PER UNIT AREA (Q).
- 2. THE HIGHER THE SPECIFIC HEAT-DENSITY OF A MATERIAL (ρc[↑]) THE SMALLER THE PEAK CONTRAST $(\Delta T \downarrow)$
- 3. THE CLOSER THE DEFECT TO THE SURFACE (d o 0) THE HIGHER THE PEAK CONTRAST ($\Delta T \rightarrow \infty$).
- 4. AS THE DEFECT DEPTH APPROACHES THE PANEL THICKNESS (d \rightarrow t) THE CONTRAST VANISHES ($\Delta T \rightarrow 0$).
- 5. FOR A GIVEN DEFECT DEPTH D, THE THICKER THE PANEL $(t \to \infty)$ THE LARGER THE CONTRAST ($\Delta T \rightarrow Q/\rho cd$).



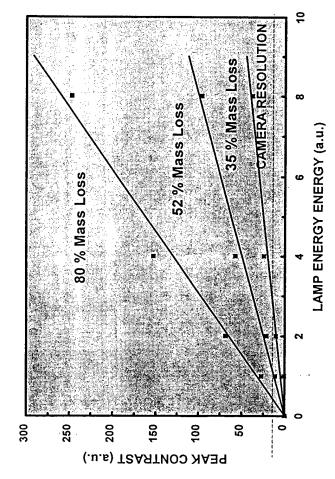
SIMPLE MODEL CORRELATION (no lateral heat flow)



CONTRAST vs DEPTH



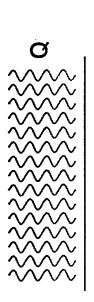
DEPTH OF RESOLUTION vs ENERGY

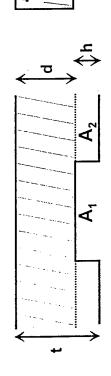


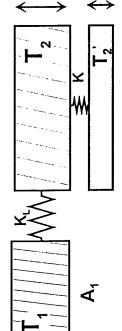


LATERAL HEAT FLOW MODEL









$$\rho \cdot A_1 \cdot d \cdot c \cdot \frac{dT_1}{dt} = k_L \cdot \frac{A_L}{R} (T_2 - T_1)$$

$$\rho \cdot A_2 \cdot d \cdot c \cdot \frac{dT_2}{dt} = k_L \cdot \frac{A_L}{R} (T_1 - T_2) + k \cdot \frac{A_2}{d+h} (T_2' - T_2)$$

$$A_2 \cdot h \cdot c \cdot \frac{dT_2}{dt} = k \cdot \frac{A_2}{d+h} (T_2 - T_2)$$

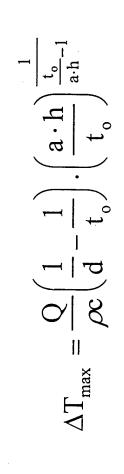






$-\frac{1+r}{d(d+h)}\frac{k}{\rho c}t$ $e^{-\frac{d(d+h)\omega}{d}}$

$$t_{max} = \frac{\rho c}{k} \frac{d \cdot t_o}{1 - a + r} \ln \frac{1 + r}{a}$$



$$a = \frac{k_L}{k} \frac{A_L}{A} \frac{d+h}{R}$$

$$r = -d$$

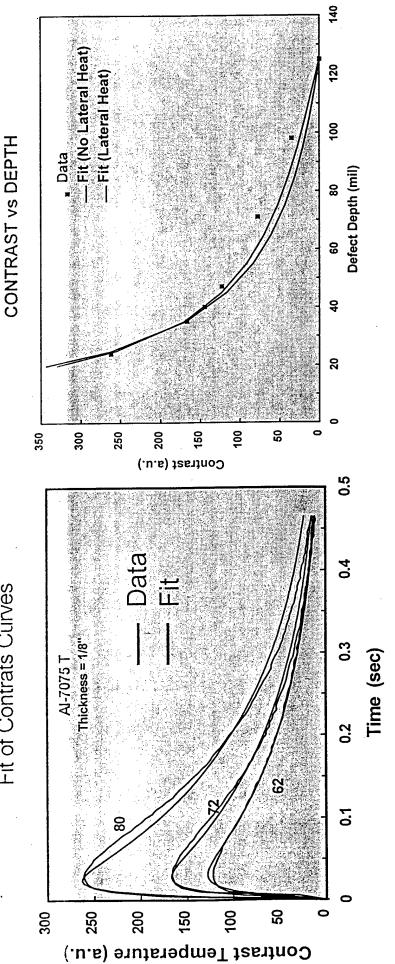
LATERAL HEAT FACTOR



THERMAL CONTRAST PREDICATIONS



Fit of Contrats Curves



$$\Gamma(t) = \frac{Q}{\rho c \cdot d \cdot (1 - a + r)} \left(e^{-\frac{a}{d(d+h)}\frac{k}{\rho c}t} - e^{-\frac{1+r}{d(d+h)}\frac{k}{\rho c}t} \right)$$

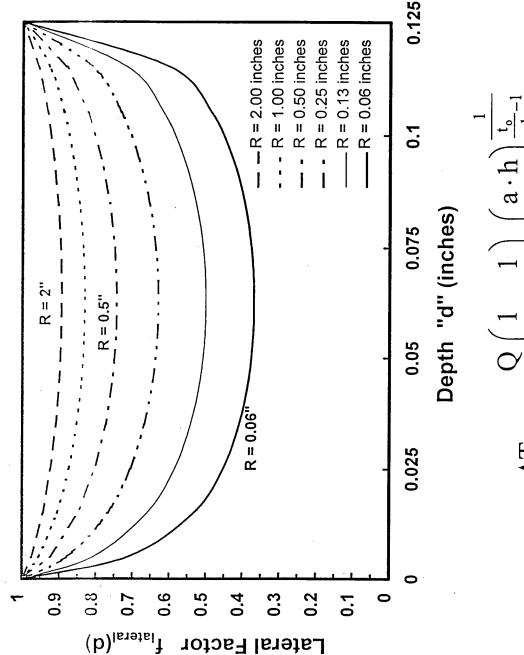
$$\Delta T_{\text{max}} = \frac{Q}{\rho c} \left(\frac{1}{d} - \frac{1}{t_s} \right) \cdot \left(\frac{a \cdot h}{t_s} \right) \frac{\frac{t_s}{t_s} - 1}{t_s}$$



(effective contact conductivity model) LATERAL HEAT FACTOR



Lateral Heat Factor





OTHER MODELING RESULTS

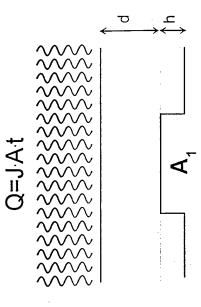


$$\Delta T(t) = \frac{Q}{\rho_1 c_1 \cdot d(r_1 + b \cdot r_1 r_2 - a_1)}.$$

$$\cdot \left(e^{-\frac{a_1}{Rd} \frac{k}{\rho_1 c_1}t} - e^{-\frac{r_1(1 + b_2)}{Rd} \frac{k}{\rho_1 c_1}t}\right)$$

$$\Delta T_{peak} = Q \cdot \left(\frac{1}{d} - \frac{1}{t_o} \right) \frac{\rho_2 c_2}{\rho_l c_1} \frac{d + h}{(d\rho_l c_1 + h\rho_2 c_2)}$$
$$\cdot \left[\frac{a_1}{r_1 (1 + b r_2)} \right]^{\frac{1}{r_1 (1 + b r_2)}}$$

$$a_1 = \frac{k_L}{k} \frac{A_d}{A}$$
 $a_2 = \frac{k_L}{k} \frac{A_h}{A}$ $r_1 = \frac{R}{d+h}$ $r_2 = \frac{d}{h}$ $b = \frac{\rho_1 c_1}{\rho_2 c_2}$



$$= \frac{J}{a \cdot k} \frac{d + h}{(1 + r)(1 + r - a)}.$$

$$\cdot \left\{ (1 + r)(1 - e^{-\frac{a \cdot k}{d(d + h)} \frac{k}{\rho^c}}) - a(1 - e^{-\frac{1 + r \cdot k}{d(d + h)} \frac{k}{\rho^c}}) \right\}$$

$$\Delta T(t \to \infty) = \frac{J}{a \cdot k} \cdot h$$

$$a = \frac{k_L}{k} \frac{A_L}{A} \frac{d+h}{R} \qquad r = \frac{d}{h}$$

$$\Delta T(t) = \frac{Q}{\rho c \cdot d(AkR + A_d k_1 d)} \left(e^{-\frac{b}{d^2} \frac{k}{\rho c}} - e^{-\frac{1}{d^2} \frac{k}{\rho c}} \right)$$

$$\Delta T_{peak} = \frac{Q}{\rho c} \frac{kR(b-1)}{d(dA_d k_1 + RAk)} \cdot [b]^{\frac{b}{1-b}}$$

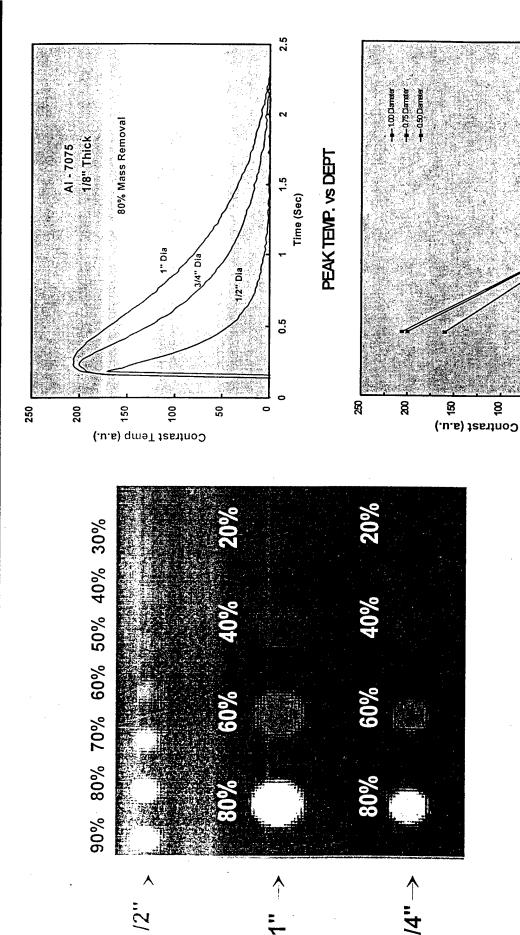
$$t_{peak} = \frac{pc}{k} \frac{d^2}{b - 1} \ln b$$
$$b = \frac{k_1}{k} \frac{A_d}{A_1} \frac{d}{R}$$



EXPERIMENTAL DATA

(80% mass removal)





5

8

8

4

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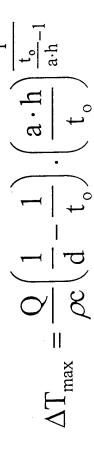
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Depth (mils)

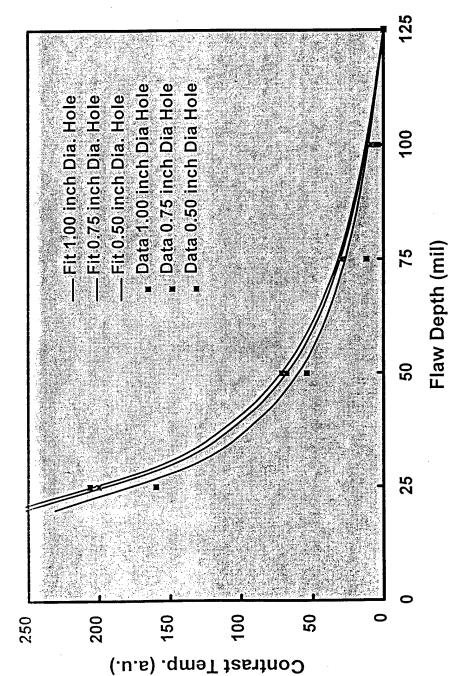


MODEL CORRELATION (effects of defect size)





Effects of Radii

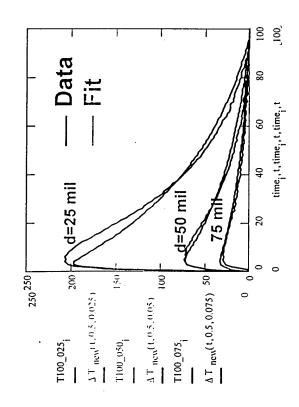




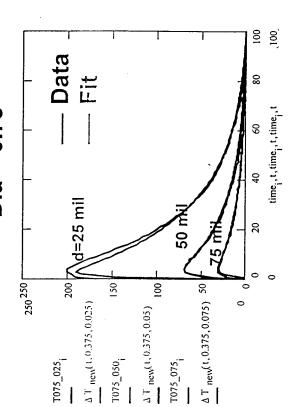
MODEL TIME-RESPONSE PREDICTIONS (varying defect sizes and locations)



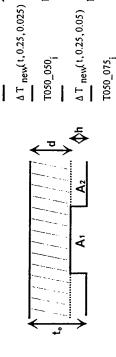




Dia = 0.75"



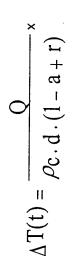
Dia = 0.50"



25 mil

150

ΔT new(1,0.25,0.075) 50



Data

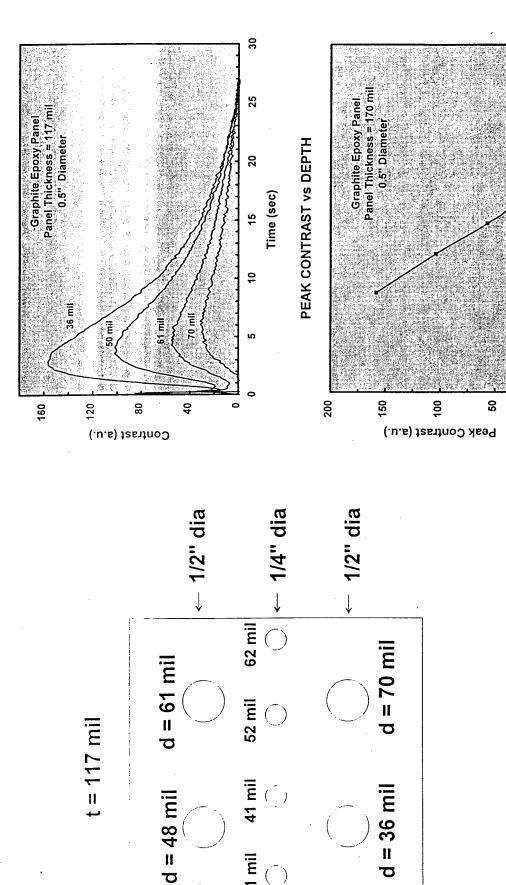
Fit

$$x \left| e^{-\frac{a}{d(d+h)}\frac{k}{\rho_c}t} - e^{-\frac{1+r}{d(d+h)}\frac{k}{\rho_c}t} \right|$$

GRAPHITE EPOXY COMPOSITE PANEL



CONTRAST vs TIME



31 mil

125

100

Depth (mil)

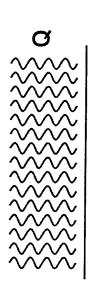
20

25



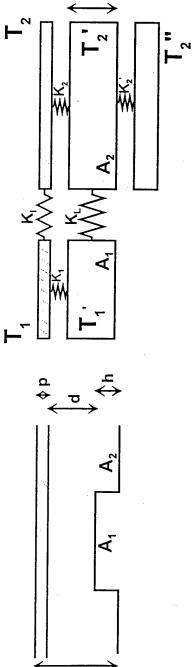
SIMPLE FINITE ELEMENT APPROXIMATION





Sample

defect



$$\rho \cdot A_1 \cdot p \cdot c \cdot \frac{dT_1}{dt} = k \cdot A_1(T_1' - T_1) + k_L \cdot A_p(T_2 - T_1)$$

$$\rho \cdot A_2 \cdot p \cdot c \cdot \frac{dT_2}{dt} = k \cdot A_2 (T_2' - T_2) + k_L \cdot A_p (T_1 - T_2)$$

$$\rho \cdot A_2 \cdot h \cdot c \cdot \frac{dT_2^n}{dt} = k \cdot A_2 (T_2^n - T_2^n)$$



FITTING RESULTS





